

SPECIFICATION

TITLE OF THE INVENTION

ELECTRIC ACTUATOR

5

TECHNICAL FIELD

The present invention relates to an electric actuator for transferring a workpiece to a working position by a transfer table driven by an electric motor and more specifically to an electric actuator in which a stepping motor driven by pulses is used as a drive source.

PRIOR ART

In this type of electric actuator for transferring various workpieces for processing, assembly, and the like, a stepping motor driven by pulses is used as a drive source in some cases. Because a rotation amount of the stepping motor can be controlled according to the number of drive pulses in stages, the stepping motor can perform an excellent function when it is used for accurately setting a transfer stroke of the workpiece or for accurately stopping the workpiece in place.

However, if the stepping motor becomes overloaded, synchronism between the stepping motor and the drive pulses is not maintained and a so-called phenomenon of

loss of synchronism is likely to be caused. Therefore,
the stepping motor is not suitable for use which requires
a constant thrust in a stopped state such as pushing the
workpiece against an object with constant pressure and
press-fitting the workpiece into a hole with constant
force.

By detecting the rotation amount of the stepping
motor, a transfer position of the work, and the like and
controlling the stepping motor such that the stepping
motor stops immediately before it becomes overloaded, it
is possible to prevent loss of synchronism and to apply a
thrust to the work in the stopped state. However, an
extremely complicated and expensive control circuit is
required.

DISCLOSURE OF THE INVENTION

It is an object of the present invention to form an
electric actuator for transferring a workpiece by using a
stepping motor as a drive source such that loss of
synchronism of the stepping motor can be prevented and
that a thrust can be applied to the workpiece in the
stopped state by providing spring means.

To achieve the above object, an electric actuator of
the invention comprises a feed screw in a shape of a
straight rod rotatably supported by a bearing member on a

base, a nut member connected to the feed screw to reciprocate in an axial direction of the feed screw by normal and reverse rotation of the feed screw, a transfer table connected to the nut member with a degree of freedom in the axial direction of the feed screw to transfer a workpiece by moving to follow the nut member, a stepping motor a rotation amount of which is controlled according to the number of drive pulses, a transmission mechanism for transmitting rotating force of the stepping motor to the feed screw, and spring means disposed between the nut member and the transfer table to elastically connect the transfer table and the nut member when the transfer table transfers the workpiece to thereby allow the nut member to overrun, to allow the stepping motor to excessively rotate in a cushioned manner, and to apply a necessary thrust to the transfer table after the transfer table reaches a transfer end of the workpiece.

In the electric actuator having the above structure, after the transfer table reaches the transfer end of the workpiece, the nut member overruns while flexing the spring means, and as a result, the stepping motor also rotates excessively in a cushioned manner. By setting the number of pulses for driving the stepping motor at a value in a range of the excessive rotation, it is possible to prevent poor synchronism due to loss of synchronism

between the stepping motor and the drive pulses. By
flection of the spring means, it is possible to apply a
thrust necessary at a transfer end to the transfer table.

According to a concrete embodiment of the invention,
the base has a sectional groove shape with an open upper
face, the feed screw extends axially in a central portion
of the groove, the nut member has a short pillar shape and
is connected to the feed screw to move in the groove in
the base, the transfer table has a sectional groove shape
and is disposed on an upper face of the base such that the
nut member is covered with the transfer table and that the
transfer table can move under a guidance of the base, and
a connecting member for connecting the transfer table and
the nut member is disposed between the transfer table and
the nut member such that the connecting member is engage
with one of the transfer table and the nut member with a
degree of freedom in the axial direction of the feed screw
and is engaged with the other with a degree of freedom in
a direction orthogonal to an axis of the feed screw.

In the invention, it is preferable that a linear
guide mechanism formed of a plurality of balls which can
roll is disposed between opposite side walls of the
transfer table and opposite groove walls of the base.

According to another concrete embodiment of the
invention, the transfer table has a spring receiver

through which the feed screw passes on at least axial one end side of the transfer table and the spring means is disposed between the spring receiver and the nut member.

In the invention, it is possible that measuring means for measuring movements of the transfer table and the nut member with respect to each other at the workpiece transfer end is provided to the transfer table. As a result, it is possible to carry out control in which the stepping motor is stopped in response to a signal from a sensor immediately before loss of synchronism when the elastic force of the spring means is set to be weaker than a thrust with which the stepping motor loses synchronism, for example.

BRIEF DESCRIPTION OF THE DRAWINGS

FIG. 1 is a front view of a first embodiment of an electric actuator according to the present invention.

FIG. 2 is a plan view of FIG. 1.

FIG. 3 is a sectional view taken along a line III-III in FIG. 2.

FIG. 4 is a sectional view taken along a line IV-IV in FIG. 3.

FIG. 5 is a sectional view taken along a line V-V in FIG. 3.

FIG. 6 is a partially cutaway exploded front view of

an essential portion in FIG. 3.

FIG. 7 is a right side sectional view of respective members in FIG. 6.

FIG. 8 is a cutaway plan view of an essential portion in FIG. 2.

FIG. 9 is a front view of a second embodiment of the electric actuator according to the invention.

FIG. 10 is a sectional view taken along a line X-X in FIG. 9.

FIG. 11 is a sectional view of a third embodiment of the electric actuator according to the invention.

FIG. 12 is a partially cutaway exploded front view of an essential portion in FIG. 11.

FIG. 13 is a sectional view of an essential portion of an initial state for explaining operation of the third embodiment.

FIG. 14 is a sectional view at a workpiece transfer end on a forward movement side for explaining operation of the third embodiment.

FIG. 15 is a sectional view at a workpiece transfer end on a rearward movement side for explaining operation of the third embodiment.

DETAILED DESCRIPTION

FIGS. 1 to 8 show a first embodiment of an electric

actuator according to the present invention. This electric actuator 1A has a base 2 having a sectional groove shape with an open upper face. The base 2 is formed to be thin and long in a groove direction, bearing members 3a and 3b are respectively mounted to axial opposite end portions of the base 2, opposite end portions of a feed screw 4 in a shape of a straight rod extending in an axial direction of the base 2 in a central portion of the groove are respectively and rotatably supported on the bearing members 3a and 3b. The feed screw 4 is formed of a ball screw, a sliding screw, or the like.

A nut member 5 in a substantially circular-cylindrical shape is screwed over the feed screw 4 and reciprocates in an axial direction of the feed screw 4 by normal and reverse rotation of the feed screw 4. A transfer table 7 for retaining and transferring a workpiece W is connected to a nut member 5 through a connecting member 8 disposed between the transfer table 7 and the nut member 5 with a degree of freedom in two direction, i.e., a direction along an axis of the feed screw 4 and a direction orthogonal to the axis. This structure will be described later in detail.

The transfer table 7 has a sectional groove shape with an open lower face and is disposed on an upper face of the base 2 such that the nut member 5 is covered with

the transfer table 7 from above and left and right
opposite side walls 7a, 7a of the transfer table 7 are
fitted with outside of left and right opposite groove
walls 2a, 2a of the base 2. The transfer table 7 can move
under a guidance of a linear guide mechanism 10 along the
groove walls 2a, 2a. The linear guide mechanism 10 is
formed by housing the large number of balls 13 in ball
holes 11 formed inside the opposite side walls 7a, 7a of
the transfer table 7 and in ball grooves 12 formed between
outer faces of the opposite side walls 7a, 7a of the
transfer table 7 and outer faces of the opposite groove
walls 2a, 2a of the base 2 such that the balls 13 can roll
in an endless row as can be seen from FIGS. 4 and 8. In
movement of the transfer table 7, these balls 13 circulate
between the ball holes 11 and the ball grooves 12 while
rolling. Two elongated holes 14 which are long in the
axial direction of the feed screw 4 are further provided
to each of the left and right opposite side walls 7a, 7a
of the transfer table 7.

On the other hand, the connecting member 8 has a pair
of left and right connecting walls 17, 17 extending
downward to be connected to the nut member 5 and
connecting screw holes 18, 18 to be connected to the
transfer table 7 and the two connecting screw holes 18 are
formed in each of left and right opposite side faces of

the connecting member 8 as can be seen from FIGS. 6 and 7.

The connecting member 8 and the nut member 5 are connected

with a degree of freedom in a vertical direction by

fitting the nut member 5 between the pair of connecting

walls 17, 17 and fitting projecting portions 17a formed on

inner side faces of the respective connecting walls 17, 17

with notch portions 5a formed in to plane shapes on

opposite outer side faces of the nut member 5 such that

the projecting portions 17a and the notch portions 5a can

move with respect to each other in the vertical direction

and are locked to each other in the axial direction. By

screwing studs 19, 19 down into the screw holes 18, 18

through the elongated holes 14 in the transfer table 7,

the transfer table 7 and the connecting member 8 are

connected through these studs 19, 19 such that the

transfer table 7 and the connecting member 8 can move with

respect to each other in the axial direction of the feed

screw 4 in a range of a length of the elongated holes 14,

14.

A spring receiver 22 through which the feed screw 4

passes is connected to a front end portion of the transfer

table 7 by a screw 27. On the other hand, a circular

spring seat 23 is in contact with a front end portion of

the nut member 5 and spring means 24 is disposed between

the spring receiver 22 and the spring seat 23. Although a

coil spring is shown as the spring means 24, the spring means 24 may be a leaf spring, an elastic body such as a sponge body, or means in which compressibility of gas is utilized.

5 In the drawings, reference numerals 25a and 25b designate stoppers mounted to a front end face of the transfer table 7 and the front bearing member 3a so as to define a forward movement end of the transfer table 7 and reference numerals 26a and 26b designate stoppers mounted to a rear end face of the transfer table 7 and the rear bearing member 3b so as to define a rearward movement end of the transfer table 7.

To a rear end portion of the base 2, a stepping motor 28 a rotation amount of which can be controlled by a drive pulse is mounted as a drive source for driving and rotating the feed screw 4. The stepping motor 28 and the feed crew 4 are connected to each other through a transmission mechanism 29. This transmission mechanism 29 connects pulleys 30a and 30b respectively mounted to an output shaft 28a of the stepping motor 28 and an end portion of the feed screw 4 to each other through a timing belt 31.

In the electric actuator 1A having the above structure, when the workpiece W is placed on the transfer table 7 in a state in which the transfer table 7 has moved

to an initial position set at the rearward movement end on
an opposite side to FIGS. 1 to 3 or an intermediate
portion, the stepping motor 28 rotates normally, the nut
member 5 moves forward by the normal rotation of the feed
screw 4, the transfer table 7 engaged with the nut member
5 through the spring means 24 also moves forward, and the
workpiece W is transferred. Then, if the transfer table 7
moves forward to a workpiece transfer end, the workpiece W
comes in contact with an object. Therefore, the transfer
table 7 stops in the position, the nut member 5 overruns
while flexing the spring means 24, and as a result, the
stepping motor 28 also rotates excessively in a cushioned
manner and stops. At this time, by setting the number of
drive pulses at a value in such a range that the stepping
motor 28 can rotate excessively and by setting elastic
force of the spring means 24 such that the stepping motor
28 does not become overloaded even if the spring means 24
is flexed in the above range, it is possible to prevent
poor synchronism due to loss of synchronism between the
stepping motor 28 and the drive pulses. Elastic force
built up by flection of the spring means 24 can apply a
thrust necessary to push the workpiece W against the
object at a transfer end to the transfer table 7.

If processing such as assembly and processing of the
workpiece W is completed and the workpiece W is removed

from the transfer table 7, the feed screw 4 moves reversely by reverse rotation of the stepping motor 28 and the nut member 5 and the transfer table 7 move rearward to the initial position. At this time, the nut member 5 and the transfer table 7 are directly engaged with each other through the connecting member 8, the studs 19, and the elongated holes 14.

FIGS. 9 and 10 show a second embodiment of the invention. An electric actuator 1B of the second embodiment is different from the first embodiment in that a flexed amount of the spring means 24 is measured by measuring movements of the transfer table 7 and the nut member 5 with respect to each other at the workpiece transfer end by measuring means 35 and that the stepping motor 28 is stopped immediately before the flexed amount of the spring means 24 exceeds a threshold value, i.e., immediately before the stepping motor 28 loses synchronism due to an overload.

In other words, hardware 36 for mounting a sensor is mounted to one side face of the transfer table 7 in this electric actuator 1B and a magnetic sensor 38 such as an automatic switch is fitted in a mounting groove 37 formed in the hardware 36. On the other hand, a permanent magnet 39 is mounted to the two studs 19, 19 on the nut member 5 side. The permanent magnet 39 and the magnetic sensor 38

are disposed in positions close to each other and the measuring means 35 is formed of the permanent magnet 39 and the magnetic sensor 38.

Here, because the studs 19, 19 are fixed to the connecting member 8 and the connecting member 8 is fixed to the nut member 5 in the axial direction of the feed screw 4 while the connecting member 8 is movable with respect to the transfer table 7, it is possible to measure changes in positions of the transfer table 7 and the nut member 5 with respect to each other, i.e., the flexed amount of the spring means 24 by detecting movements of the transfer table 7 and the studs 19, 19 with respect to each other as described above.

In use of such measuring means 35, it is possible to set the elastic force of the spring means 24 to be weaker than the thrust with which the stepping motor 28 loses synchronism.

Because structures and operations of the second embodiment other than those described above are substantially similar to those of the first embodiment, the main similar components are provided with the similar reference numerals to omit description of them so as to avoid repetition of description.

The measuring means 35 is not limited to one formed of the permanent magnet 39 and the magnetic sensor 38 as

described above but may be one formed of a magnetic scale
formed by disposing magnetic portions and non-magnetic
portions at constant intervals and a magnetic sensor for
reading a scale of the magnetic scale. In this case,
5 which of the magnetic scale and the magnetic sensor is
mounted to the transfer table 7 or the nut member 5 may be
arbitrarily determined.

FIGS. 11 to 15 show a third embodiment of the
invention. An electric actuator 1C of the third
embodiment is different from the above first embodiment in
that the nut member 5 and the transfer table 7 are
elastically connected by respective spring means 24a and
24b in both strokes of forward and rearward movements by
respectively providing the spring means 24a and 24b
between front end portions and between rear end portions
of the nut member 5 and the transfer table 7 in the third
embodiment while the nut member 5 and the transfer table 7
are elastically connected by the spring means 24 only in
the forward movement stroke by providing the one spring
20 means 24 between the front end portion of the nut member 5
and the front end portion of the transfer table 7 in the
first embodiment.

In other words, as shown in FIGS. 11 and 12, spring
receivers 22a and 22b through which the feed screw 4 pass
25 are respectively connected to the front and rear opposite

end portions of the transfer table 7, annular spring seats 23a and 23b are disposed at the front and rear opposite end portions of the nut member 5 such that the spring seats 23a and 23b can come in contact with and separate from the nut member 5, and the spring means 24a and 24b are respectively disposed between the respective spring receivers 22a and 22b and the spring seats 23a and 23b. A base end portion of a sleeve 41 for restricting a separating distance between the spring seat 23a and the spring receiver 22a (i.e., an expanding length of the spring means 24a) is screwed to the front spring seat 23a, a tip end of the sleeve 41 is fitted for sliding in a screw insertion hole 42 inside the spring receiver 22a, a flange-shaped locking portion 41a is formed at the tip end of the sleeve 41, and a small-diameter portion 42a to which the locking portion 41a is locked is formed at an end portion of the screw insertion hole 42. As shown in FIG. 15, when the spring seat 23a separates from the spring receiver 22a by a maximum distance, the locking portion 41a at the tip end of the sleeve 41 is locked to the small-diameter portion 42a of the screw insertion hole 42 and further separating is restricted. There is such a relationship between the two spring means 24a and 24b that the front spring means 24a has larger elastic force than the rear spring means 24b.

Because structures of the third embodiment other than those described above are substantially similar to those of the first embodiment, the main similar components are provided with the similar reference numerals to omit description of them.

In the electric actuator 1C of the third embodiment having the above structure, the transfer table 7 is in an intermediate initial position shown in FIG. 13 in a non-operating state. At this time, because the locking portion 41a at the tip end of the sleeve 41 is locked to the small-diameter portion 42a of the screw insertion hole 42, the front spring seat 23a and the spring receiver 22a are separated from each other by the maximum distance. On the other hand, the rear spring means 24b is in a slightly compressed state.

If the workpiece W is placed on the transfer table 7 in this state and the stepping motor 28 rotates normally, the nut member 5 moves forward by normal rotation of the feed screw 4, the transfer table 7 is engaged with the nut member 5 through the front spring means 24a to thereby move forward, and the workpiece W is transferred. Then, if the transfer table 7 moves forward to the transfer end of the workpiece W as shown in FIG. 14, the workpiece W comes in contact with the object. Therefore, the transfer table 7 stops in the position, the nut member 5 overruns

while flexing the front spring means 24a, and as a result,
the stepping motor 28 also rotates excessively and stops
in a cushioned manner. At this time, by setting the
number of drive pulses at a value in such a range that the
stepping motor 28 can rotate excessively and by setting
elastic force of the spring means 24a such that the
stepping motor 28 does not become overloaded even if the
spring means 24a is flexed in the above range similarly to
the first embodiment, it is possible to prevent poor
synchronism due to loss of synchronism between the
stepping motor 28 and the drive pulses. Elastic force
built up by flection of the spring means 24a can apply a
thrust necessary to push the workpiece W against the
object at the transfer end of the workpiece W to the
transfer table 7.

If processing such as assembly and processing of the
workpiece W is completed and the workpiece W is removed
from the transfer table 7, the feed screw 4 moves
reversely by reverse rotation of the stepping motor 28 and
the nut member 5 and the transfer table 7 move rearward to
the initial position.

If the stepping motor 28 rotates reversely after the
workpiece W is placed on the transfer table 7 in the
initial position, the nut member 5 moves rearward by
reverse rotation of the feed screw 4, the transfer table 7

is engaged with the nut member 5 through the rear spring means 24b to thereby move rearward, and the workpiece W is transferred reversely. Then, if the transfer table 7 moves to the transfer end in a direction of the rearward movement as shown in FIG. 15, the workpiece W comes in contact with the object. Therefore, the transfer table 7 stops at the position, the nut member 5 overruns while flexing the rear spring means 24b, and as a result, the stepping motor 28 also rotates excessively and stops in a cushioned manner. At this time, because movement of the spring seat 23a is restricted by the sleeve 41, the front spring means 24a does not expand further than the initial state. Therefore, the front spring means 24a does not affect in the rearward movement stroke. Thus, at the rearward movement stroke end, it is also possible to prevent poor synchronism due to loss of synchronism of the stepping motor 28 similarly to the case of the forward movement stroke end and it is possible to obtain the thrust necessary to push the workpiece W against the object by flection of the spring means 24b.

In the third embodiment, by measuring movements of the transfer table 7 and the nut member 5 with respect to each other at at least one of the forward and rearward transfer stroke ends, the stepping motor 28 can be stopped immediately before the stepping motor 28 loses synchronism

due to the overload similarly to the second embodiment.

Although the connecting member 8 for connecting the transfer table 7 and the nut member 5 is engaged with the transfer table 7 with a degree of freedom in the axial direction of the feed screw 4 and is engaged with the nut member 5 with a degree of freedom in the direction orthogonal to the axis of the feed screw 4 in the above respective embodiments, it is also possible that the connecting member 8 is engaged with the nut member 5 with a degree of freedom in the axial direction of the feed screw 4 and is engaged with the transfer table 7 with a degree of freedom in the direction orthogonal to the axis of the feed screw 4 on the contrary.

As described above, according to the invention, by providing the spring means to the electric actuator for transferring the workpiece by using the stepping motor as the drive source, it is possible to prevent loss of synchronism of the stepping motor and to apply the thrust to the workpiece W in the stopped state.